

2.615 HW #8 Soln

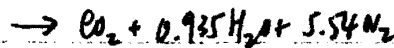
1) (Prob. 11.3 of Text)

$$\text{Exhaust mass flow rate} = \dot{m}_f (1 + A/F)$$

$$\text{To change mass fraction to mole fraction } \tilde{x}_i = x_i \frac{\bar{W}}{w_i}$$

where \bar{W} is the av. exhaust M.W.; w_i is the M.W. of species i

For gasoline stoichiometric combustion: $\text{C}_8\text{H}_{18} + 14.7(\text{O}_2 + 3.773\text{N}_2)$



$$\text{Av. Mol. M.W.} = \frac{44 + 0.935 \times 18 + 5.54 \times 28.12}{1 + 0.935 + 5.54} = 28.96 \approx 29$$

$$\begin{aligned} \text{Exhaust species mole fractions: } \tilde{x}_{\text{NO}_2} &= \frac{\dot{m}_{\text{NO}_2}}{\dot{m}_f (1 + A/F)} \frac{\bar{W}}{w_{\text{NO}_2}} = \frac{1.5}{100 \times (1 + 14.6)} \frac{29}{46} \\ &= 5.05 \times 10^{-4} = \underline{505 \text{ ppm}} \end{aligned}$$

$$\tilde{x}_{\text{HC}} = \frac{2}{100 \times (1 + 14.6)} \frac{29}{13.85} = 2.24 \times 10^{-3} = \underline{2240 \text{ ppm}}$$

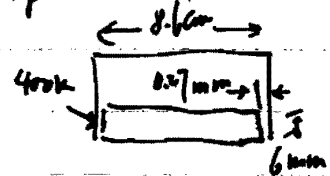
$$\tilde{x}_{\text{CO}} = \frac{20}{100 \times (1 + 14.6)} \frac{29}{28} = 1.11 \times 10^{-2} = \underline{11000 \text{ ppm or } 1.11\%}$$

2) (11-9 Text) (a) Mass of charge = $m_a + m_f + m_p = m_a (1 + F/A)$; assume $x_r = 10\%$

$$m_a = P_{\text{atm}} V_0 \eta_v = \left(\frac{10^5}{287 \times 300} \right) (500 \times 10^{-6}) \times 0.8 \times 0.9 = 4.6 \times 10^{-4} \text{ kg} = 0.46 \text{ g}$$

$$m_{\text{charge}} = \frac{0.46 (1 + 14.6)}{1 - 0.1} \text{ g} = 0.55 \text{ g}$$

$$V_{\text{exh.}} = \pi D \delta L \times 1.2 = \pi \times 8.6 \times 0.02 \times 0.6 \times 1.2 = 0.53 \text{ cc}$$



molecular wt of both burned and unburned gas are about the same; $\bar{W} \approx 29$

$$\text{Peak mass in groove} = \frac{P_{\text{peak}} V_{\text{exh.}}}{(\bar{R}/\bar{W}) T_{\text{exh.}}} = \frac{3 \times 10^6 \times 0.53 \times 10^{-6}}{8314/29 \times 400} \text{ kg} = 1.37 \times 10^{-5} \text{ kg} = 1.37 \times 10^{-2} \text{ g}$$

$$\frac{m_{\text{exh.}}}{m_{\text{charge}}} = \frac{1.37 \times 10^{-2}}{0.55} = 2.5\%$$

(b) Unburned gas in exhaust

$$m_u = m_{cr} \underbrace{\frac{1}{3}}_{\substack{\text{unburned fraction} \\ \text{in mixture}}} \cdot \underbrace{\left(1 - \frac{1}{2}\right) \left(1 - \frac{1}{7}\right)}_{\text{oxidation}} = 1.37 \times \frac{2}{3} \times \frac{1}{2} \times \frac{2}{3} = 3.0 \times 10^{-3} \text{ g}$$

unburned fuel (HC) in exhaust

$$m_{HC} = m_u \left(\frac{1}{1 + \phi} \right) = 3 \times 10^{-3} \times \frac{1}{15.6} = 1.95 \times 10^{-4} \text{ g}$$

unburned mole fraction of HC, $\tilde{x}_{HC} = \frac{m_{HC}}{m_u (1 + F/A)} \frac{W}{W_{HC}}$

for $W_{HC} = 13.85$ ($F/A = 1.85$) $\tilde{x}_{HC} = \frac{1.95 \times 10^{-4}}{0.46 \times (1 + \frac{1}{15.6})} \frac{29}{13.85} = 8.3 \times 10^{-4} = 830 \text{ ppm}$

(c) Brake specific HC emission

$$BSHC = \frac{m_{HC}}{m_f} \frac{m_f}{W_{brake}} = \left(\frac{1.95 \times 10^{-4}}{0.46 \times \frac{1}{15.6}} \right) \frac{29}{13.85} = \underline{\underline{1.86 \text{ g/kWh}}}$$

3) (11-10 of text)

$$\frac{d[N_2]}{dt} \Big|_0 = 2k_1^+ [N_2]_e [O]_e \quad \text{for } [] \text{ in mol/cc}$$

since $[N_2] = \eta \tilde{x}_{N_2}$ where η is the total mol/cc

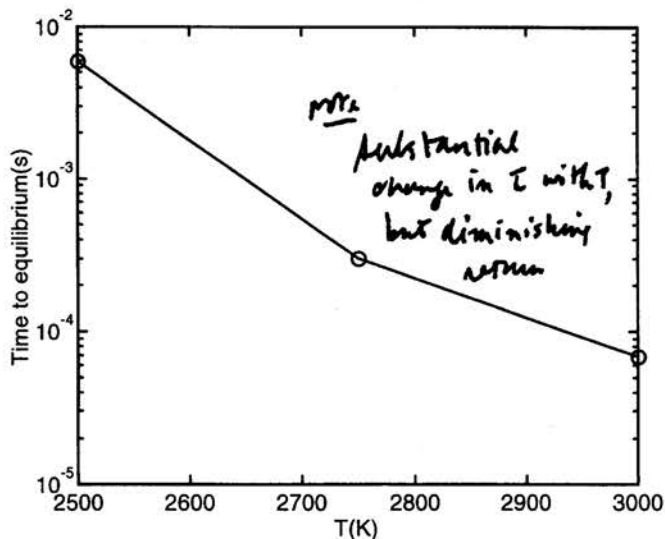
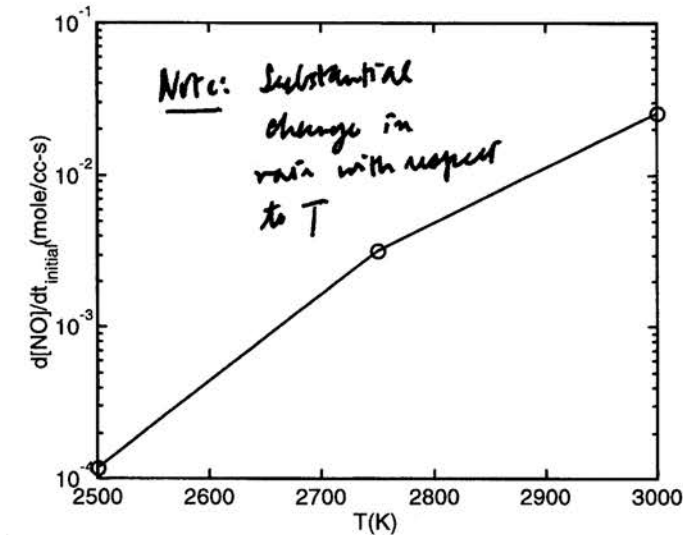
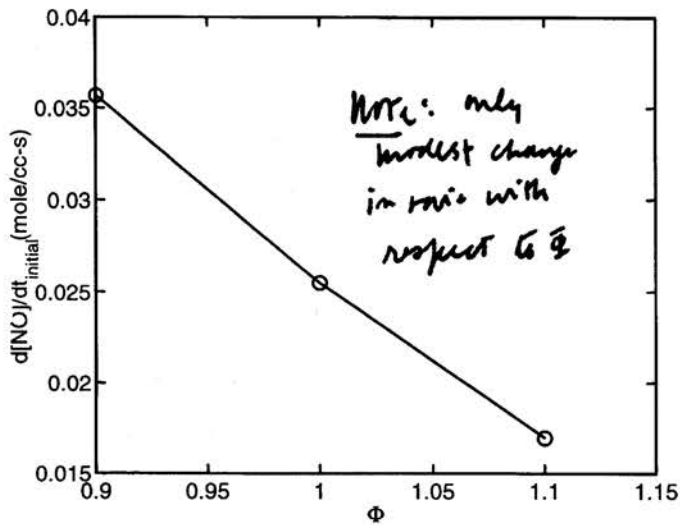
and $\eta = \left(\frac{p}{RT} \right) \text{ kmol/m}^3 = \left(\frac{p}{RT} \right) \frac{\times 10^3}{10^6} \text{ mol/cc} = \left(\frac{p}{RT} \right) \times 10^{-3} \text{ mol/cc}$

$$\frac{d[N_2]}{dt} \Big|_0 = 2k_1^+ \left(\frac{p}{RT} \times 10^{-3} \right)^2 \tilde{x}_{N_2, e} \tilde{x}_{O, e} \quad ; \quad \tau_{N_2} = \frac{\left(\frac{p}{RT} \times 10^{-3} \right) \tilde{x}_{N_2, e}}{[N_2]_0}$$

(mol/cc-s)

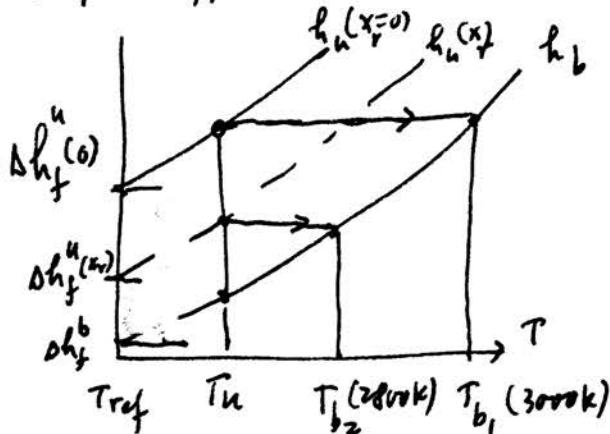
$k_1^+ = 7.6 \times 10^{11} \exp\left(\frac{-38200}{T}\right)$ where T is in K

ϕ	T	\tilde{x}_{O_2}	\tilde{x}_{N_2}	$[N_2]_0$ mol/cc-s	T	\tilde{x}_{O_2}	\tilde{x}_{N_2}	$\tilde{x}_{NO, eq}$	$[N_2]_0$	τ (s)
0.9	↑	2.1×10^{-3}	↑	3.57×10^{-2}	2500	6×10^{-5}	↑	2.6×10^{-3}	1.17×10^{-3}	5.4×10^{-3}
1.0	3000	1.5×10^{-3}	0.73	2.55×10^{-2}	2750	5×10^{-4}	0.73	4×10^{-3}	3.2×10^{-3}	3×10^{-4}
1.1	↓	1×10^{-3}	↓	1.7×10^{-2}	3000	1.5×10^{-3}	↓	8×10^{-3}	2.6×10^{-2}	6.8×10^{-5}



Factor of 4 reduction in $[NO] \Rightarrow [NO]$ reduces from 2.55×10^{-2} mol/cc-s to 0.64×10^{-2} mol/cc-s. From the graph, Temperature has to drop from 3000K to 2800K

Graphically, the x_r needed is illustrated as



Combustion at prevailing pressure sensible part $h_b = h_u$, Thus

$$\rightarrow h_s^b(T_{b1}) + \Delta h_f^b = h_s^u(T_u) + \Delta h_f^u(x_r=0)$$

$$\rightarrow h_s^b(T_{b2}) + \Delta h_f^b = h_s^u(T_u) + \Delta h_f^u(x_r)$$

$$h_s^b(T_{b1}) - h_s^b(T_{b2}) = \Delta h_f^u(x_r=0) - \Delta h_f^u(x_r)$$

Using Eq 4.32, at $q=1$

$$\Delta h_f^u(x_r=0) - \Delta h_f^u(x_r) = x_r \cdot 2951 \text{ kJ/mole air} \\ = x_r \cdot 2951 \times \frac{15.6}{14.6} \text{ kJ/mole fuel/air mixture}$$

From Fig 4-17, $c_{p,b} \approx 2000 \text{ J/kgK}$

Thus

$$x_r = \frac{c_{p,b}(T_{b1} - T_{b2})}{2951 \times 10^3 \times \frac{15.6}{14.6}} = \frac{2000(200)}{2951 \times 10^3 \times \frac{15.6}{14.6}}$$

$$= \underline{\underline{0.13}}$$

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2.61 Internal Combustion Engines
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