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2.004 Dynamics and Control II
Spring 2008

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MASSACHUSETTS INSTITUTE OF TECHNOLOGY
 DEPARTMENT OF MECHANICAL ENGINEERING

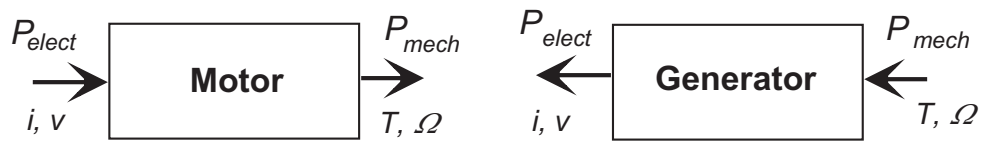
2.004 *Dynamics and Control II*
 Spring Term 2008

Solution of Problem Set 2

Assigned: Feb. 15, 2008

Due: Feb. 22, 2008

Problem 1:



(a) Assuming an ideal dc motor:

$$Power_{in} = Power_{elect} = v_b i_m = K_v \Omega_m i_m$$

$$Power_{out} = Power_{mech} = T_m \Omega_m = K_m i_m \Omega_m$$

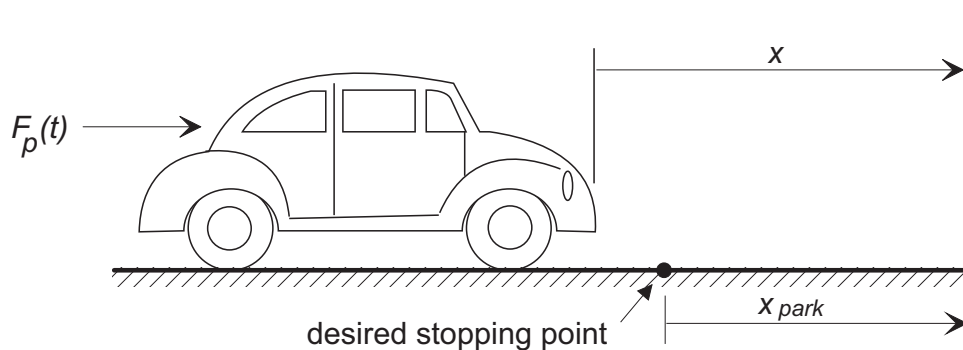
$$Power_{in} = Power_{out} \Rightarrow K_v \Omega_m i_m = K_m i_m \Omega_m \Rightarrow K_v = K_m$$

(b) Using the data sheet:

$$K_m = 30.2 \frac{mNm}{A} = 30.2 \times 10^{-3} \frac{Nm}{A}$$

$$K_v = \frac{1}{317 \frac{rpm}{V}} = \frac{1}{317} \frac{V}{rpm} = \frac{1}{317 \times \frac{2\pi}{60} \frac{rad}{s}} = 30.1 \times 10^{-3} \frac{V}{rad/s}$$

Problem 2:



Using an approach similar to that we used in class (with the same basic mass/friction linear model):

(a)

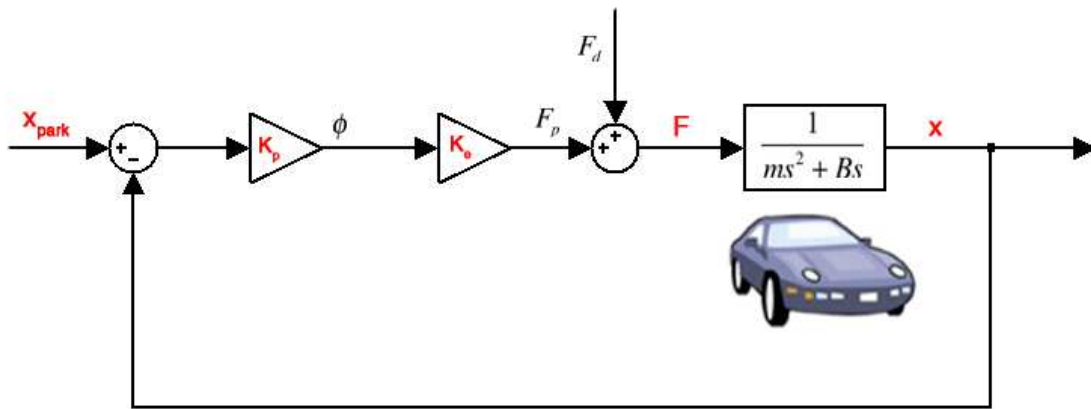
$$m \frac{dv}{dt} + Bv = F_p(t) + F_d(t)$$

Ignore F_d , assume $F_p(t) = K_e \phi(t)$ and employ $v = \frac{dx}{dt}$:

$$m \frac{d^2x}{dt^2} + B \frac{dx}{dt} = K_e \phi(t)$$

$$\frac{X(s)}{\Phi(s)} = \frac{K_e}{ms^2 + Bs}$$

(b-c)



(d)

$$m \frac{d^2x}{dt^2} + B \frac{dx}{dt} = K_e K_p (x_{park} - x)$$

In the steady state, $\frac{dx}{dt} = 0$ hence:

$$x_{s.s.} = x_{park}$$

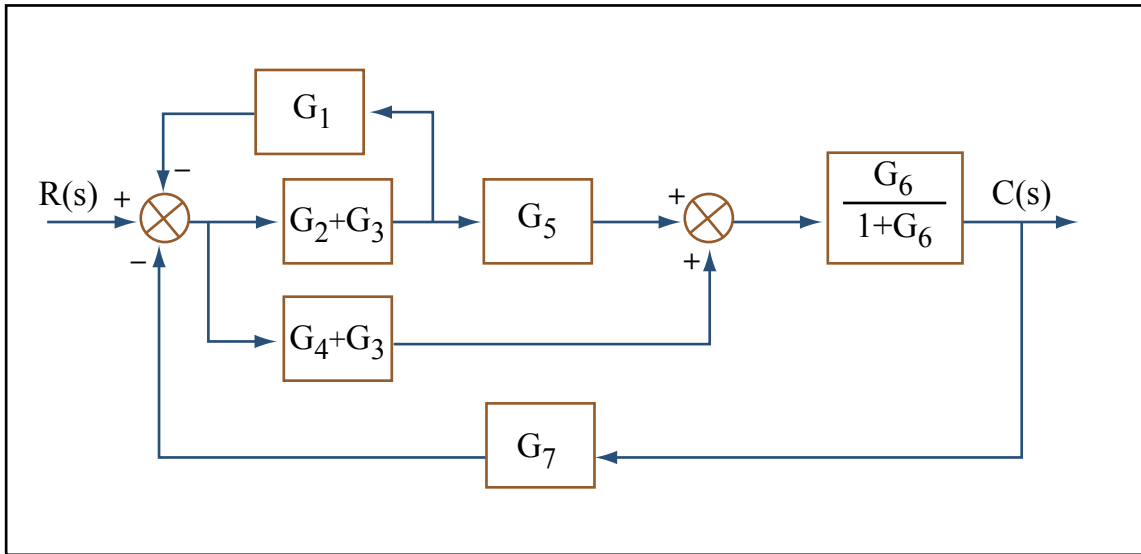
(e)

$$\begin{aligned} \text{Position Control: } m \frac{d^2x}{dt^2} + B \frac{dx}{dt} &= K_e K_p (x_{park} - x) \\ \text{Velocity Control: } m \frac{dv}{dt} + Bv &= K_e K_p (v_{desired} - v) \end{aligned}$$

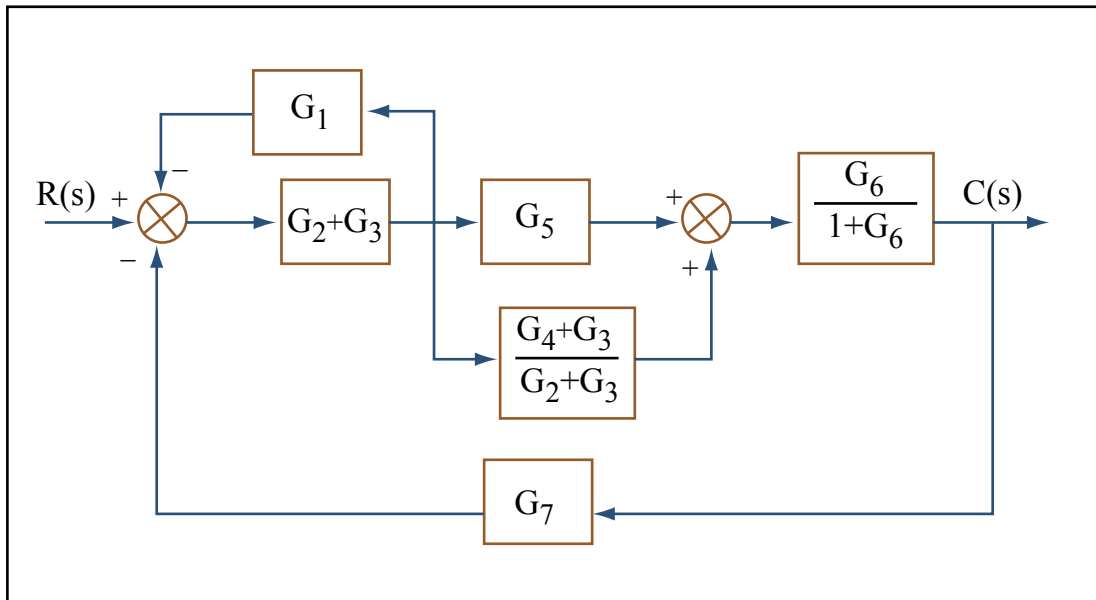
In the position control $v_{s.s.} = 0$, all the left terms of the equation are zero and a zero error can be held. On the other hand for the velocity control, $\frac{dv}{dt}|_{s.s.} = a_{s.s.} = 0$ but $v_{s.s.} \neq 0$, the drag force is not zero, and then $v_{s.s.} \neq v_{desired}$.

Problem 3: Nise, Chapter 5, Problem 4.

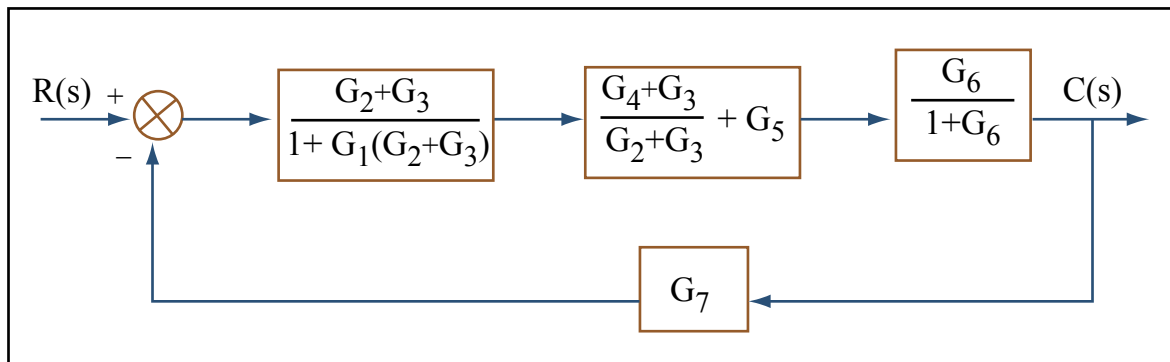
- (1) Split G_3 and combine it with G_2 and G_4 . Also use feedback formula on G_6 loop:



- (2) Push $G_2 + G_3$ to the left past the pickoff point:



- (3) Using the feedback formula and combining parallel blocks:

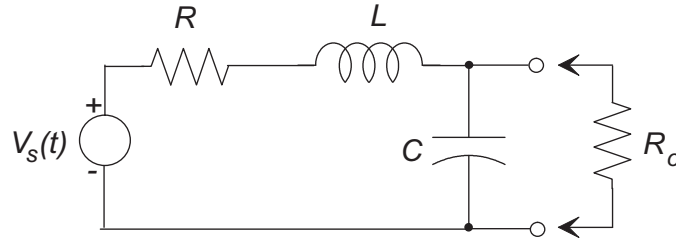


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(4) Multiplying the blocks of the forward path and applying the feedback formula:

$$T(s) = \frac{G_6 G_4 + G_6 G_3 + G_6 G_5 G_3 + G_6 G_5 G_2}{1 + G_6 + G_3 G_1 + G_2 G_1 + G_7 G_6 G_4 + G_7 G_6 G_3 + G_6 G_3 G_1 + G_6 G_2 G_1 + G_7 G_6 G_5 G_3 + G_7 G_6 G_5 G_2}$$

Problem 4:



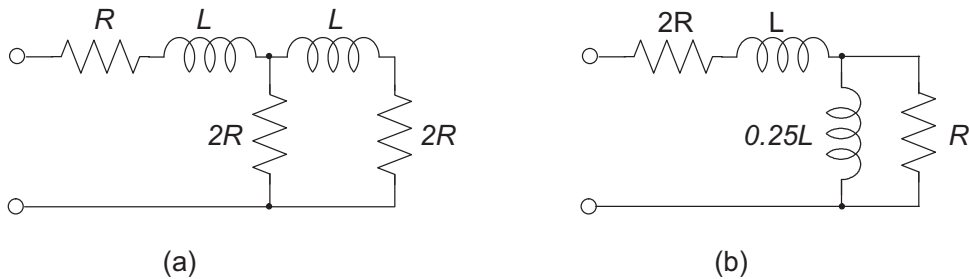
$$Z_o = R_o \parallel \frac{1}{Cs} = \frac{R_o \frac{1}{Cs}}{R_o + \frac{1}{Cs}} = \frac{R_o}{R_o Cs + 1}$$

$$Z_s = R + Ls + Z_o = R + Ls + \frac{R_o}{R_o Cs + 1}$$

Voltage Division:

$$\begin{aligned} G(s) &= \frac{V_o}{V_s} = \frac{Z_o}{Z_s} = \frac{\frac{R_o}{R_o Cs + 1}}{R + Ls + \frac{R_o}{R_o Cs + 1}} \\ &= \frac{R_o}{LR_o Cs^2 + (L + RR_o C)s + (R + R_o)} \end{aligned}$$

Problem 5: Input impedance does not characterize a system uniquely. In other words, systems with different characteristics, like below examples, can have the same input impedance.



(a)

$$\begin{aligned} Z_a &= R + Ls + 2R \parallel (Ls + 2R) \\ &= R + Ls + \frac{2R(Ls + 2R)}{2R + (Ls + 2R)} \\ &= R + Ls + \frac{2RLs + 4R^2}{4R + Ls} \\ &= R + Ls + \frac{0.5RLs + R^2}{R + 0.25Ls} \end{aligned}$$

(b)

$$\begin{aligned}Z_b &= 2R + L_s + R \parallel (0.25L_s) \\&= 2R + L_s + \frac{R(0.25L_s)}{R + 0.25L_s} \\&= R + L_s + R + \frac{R(0.25L_s)}{R + 0.25L_s} \\&= R + L_s + \frac{0.5RL_s + R^2}{R + 0.25L_s} \\&= Z_a\end{aligned}$$