

LECTURE # 24

1.060 ENGINEERING MECHANICS II

UNIFORM FLOW COMPUTATIONS

Geometry of Prismatic Channel, i.e.

$A(h)$, $P(h)$, $R_h = A/P = R_h(h)$, "known"

Channel Slope S_0 "known"

Channel Roughness ϵ (or Manning's n) "known"

from Table 10.1 or $n = 0.038 \epsilon^{1/6}$ (SI)

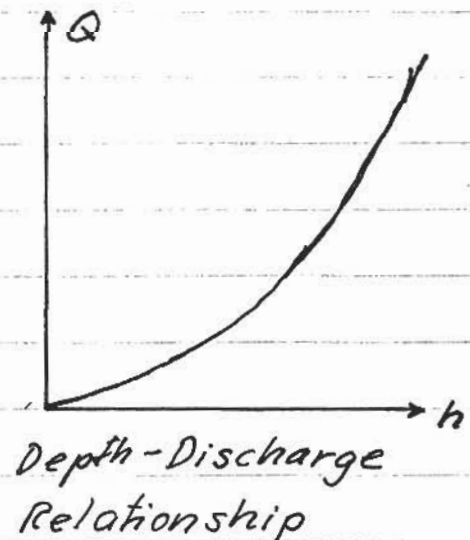
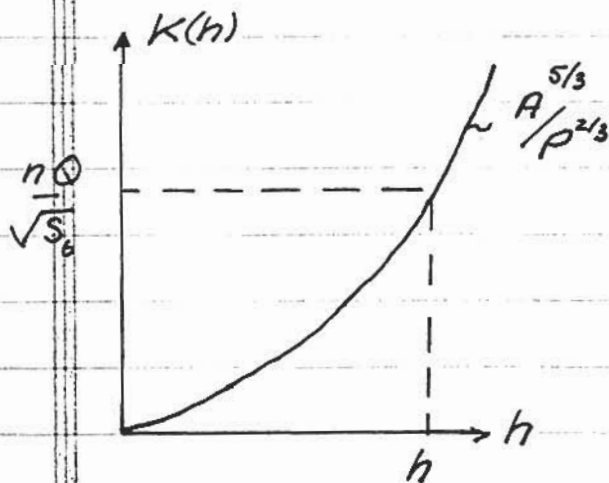
Type 1 Problem: h specified \Rightarrow Find Q

$$Q = V \cdot A = \frac{1}{n} R_h^{2/3}(h) A(h) \sqrt{S_0} = \frac{1}{n} \frac{[A(h)]^{5/3}}{[P(h)]^{2/3}} S_0^{1/2}$$

Not a hard problem.

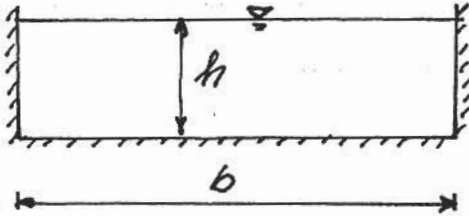
Type 2 Problem: Q specified \Rightarrow Find stage, i.e. h

$$Q = VA = \frac{1}{n} \frac{A^{5/3}}{P^{2/3}} \sqrt{S_0} \Rightarrow \frac{A^{5/3}}{P^{2/3}} = K(h) = \frac{nQ}{\sqrt{S_0}}$$



Simple Geometry (Iterative Solution)

Rectangular Channel



$$A = bh, \quad P = b + 2h$$

$$R_h = A/P = bh/(b+2h) = h/(1+2h/b)$$

Very Wide Rec. Channel $h \ll b$

$$R_h \approx h$$

$$Q = AV = \frac{1}{n} R_h^{2/3} A \sqrt{S_0} = \frac{1}{n} \frac{h^{2/3}}{(1+2h/b)^{2/3}} hb \sqrt{S_0}$$

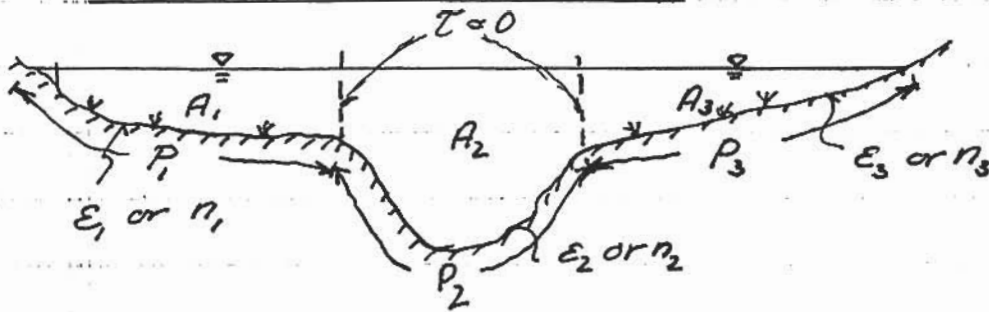
$$\frac{Qn}{\sqrt{S_0} b} = \frac{h^{5/3}}{(1+2h/b)^{2/3}} \Rightarrow h = \underbrace{\left(\frac{Qn}{b\sqrt{S_0}} \right)^{3/5}}_K (1+2\frac{h}{b})^{2/5}$$

Solve by iteration:

$$h^{(n+1)} = K \left(1 + 2\frac{h^{(n)}}{b} \right)^{2/5}$$

starting with $h^{(0)} = 0$.

COMPOSITE CHANNELS

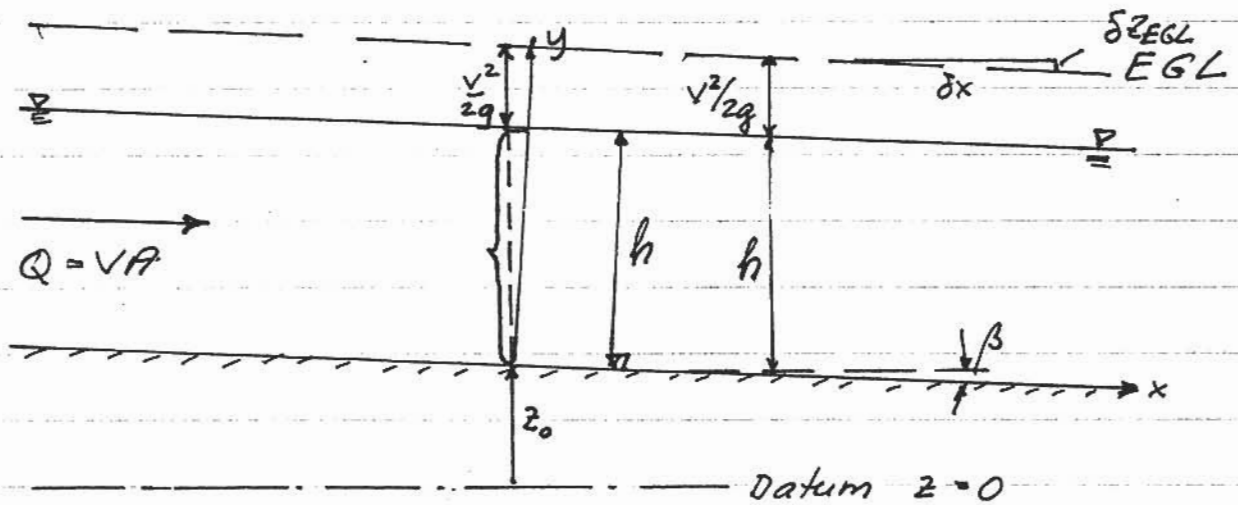


No shear stress along fluid-air and fluid-fluid interfaces.

$$Q = \sum Q_n = \sum \frac{1}{n_n} A_n \left(\frac{A_n}{P_n} \right)^{2/3} \sqrt{S_0}$$

ENERGY AND HYDRAULIC GRADE LINES

Uniform, Steady Flow



z_{EGL} = vertical distance of Energy Grade Line (Total Head Line) above datum = $H = z + \frac{p}{\rho g} + \frac{v^2}{2g}$,

BERNOULLI

Since flow is steady, uniform and well behaved $p + \rho g z = \text{constant} \perp$ streamlines. Within fluid we have: $z = z_0 + y \cos \beta$, p_0

$p + \rho g z = p + \rho g (z_0 + y \cos \beta) = \text{const} = \rho g (z_0 + h \cos \beta)$
since $p = 0$ at free surface. So, within fluid

$$\frac{p}{\rho g} + z = \frac{p + \rho g z}{\rho g} = z_0 + h \cos \beta = z_{HGL}$$

So,

$$z_{EGL} = H = z_0 + h \cos \beta + \frac{v^2}{2g} = z_0 + h \cos \beta + \frac{Q^2}{2gA^2}$$

Since, in most cases $\cos \beta \approx 1.000..$,

$$z_{EGL} = H = h + \frac{Q^2}{2gA^2} + z_0$$

EGL is parallel to bottom

In a uniform steady flow

HGL = hydraulic grade line

has

$$Z_{HGL} = z_0 + h \cos \beta$$

i.e. it is parallel to the bottom and a vertical distance $h \cos \beta$ above it.

Since, in most cases $\cos \beta \approx 1$, the hydraulic grade line is "identical" to the free surface.

EGL = energy grade line

has

$$H = Z_{EGL} = z_0 + h \cos \beta + \frac{V^2}{2g} = z_0 + h \cos \beta + \frac{Q^2}{2gA^2}$$

and is located a distance of $V^2/2g = Q^2/2gA^2 =$

The velocity head above the HGL, i.e. it too is parallel to the bottom.

$$\text{Slope of bottom} = -\frac{\partial z_0}{\partial x} = \sin \beta = S_0 =$$

$$\text{Slope of EGL} = -\frac{\partial H}{\partial x} = S_f = S_0$$

is valid for uniform, steady flow.

From the generalized pipe flow expression

$$-\frac{\partial H}{\partial x} = \frac{\tau_s P}{\rho g A} = \frac{\tau_s}{\rho g R_h} = S_f$$

we have

$$S_f = \frac{f}{8g} \frac{P}{A^3} Q^2 \quad (\text{Darcy-Weisbach})$$

for Darcy-Weisbach friction, $\tau_s = \frac{1}{8} \rho f V^2$.
For Chezy's Equation, $C = \sqrt{8g/f}$, we

$$S_f = \frac{1}{C^2} \frac{P}{A^3} Q^2 \quad (\text{Chezy's Equation})$$

where we refer to this equation as Chezy's Equation only when $C = \text{Chezy Coefficient}$ is treated as a constant

Finally, we have for Manning's Equation

$$S_f = n^2 \frac{P^{4/3}}{A^{10/3}} Q^2 \quad (\text{Manning's Equation})$$

i.e. the bottom shear stress is, in terms of Manning's "n" given by

$$\tau_s = \rho g n^2 \left(\frac{P}{A}\right)^{1/3} V^2 = \rho g n^2 R_h^{-1/3} V^2$$

For uniform, steady flow $S_f = S_0$ and the above formulae for S_f reduce to the equations we use to determine uniform, steady flow characteristics.